

1 Locally linear systems

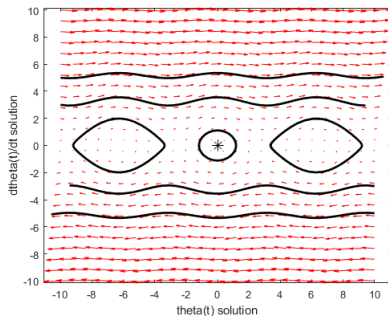


Figure: Phase portrait and solutions for undamped oscillating pendulum.

We will study systems

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where the components of $\mathbf{f}$ are C^1 functions so that we are able to Taylor expand them.

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where the components of $\mathbf{f}$ are C^1 functions so that we are able to Taylor expand them. The following system

$$\dot{\mathbf{x}} = \mathbf{A}\mathbf{x} + \mathbf{g}(\mathbf{x})$$

is called *locally linear* around a critical point $\mathbf{x}_0$ if

$$\frac{\|\mathbf{g}(\mathbf{x})\|}{\|\mathbf{x}\|} \rightarrow 0 \text{ as } \mathbf{x} \rightarrow \mathbf{x}_0.$$

Presenting the method: oscillating pendulum

Consider the following oscillating pendulum: a mass m is attached to one end of a rigid, but weightless, rod of length L which hangs from the pivot point.

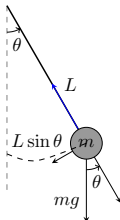


Figure: oscillating pendulum

Presenting the method: oscillating pendulum

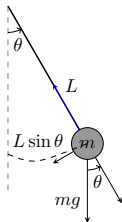


Figure: oscillating pendulum

The gravitational force mg acts downward and the damping force $c|\frac{d\theta}{dt}|$ is always opposite to the direction of motion. A rotational analog of Newton's second law of motion might be written in terms of torques:

$$mg \cdot L \sin(\theta) + \frac{d\theta}{dt} \cdot L + m \frac{d^2\theta}{dt^2} L^2 = 0 \Rightarrow \frac{d^2\theta}{dt^2} + \gamma \frac{d\theta}{dt} + \omega^2 \sin(\theta) = 0.$$

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This is a nonhomogeneous second order equation, but we can also view it as a system of equations by letting $x := \theta$ and $y := \frac{d\theta}{dt}$:

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$$\frac{dx}{dt} = y, \quad \frac{dy}{dt} = -\gamma y - \omega^2 \sin(x),$$

where γ is called the damping constant and as in the spring problem it is responsible for removing energy. This is an autonomous system.

Presenting the method: oscillating pendulum

(1) we find the critical points:

$$\frac{dx}{dt} = 0, \frac{dy}{dt} = 0 \Rightarrow y = 0, \sin(x) = 0 \Rightarrow (k\pi, 0) \text{ for } k \in \mathbb{Z}.$$

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(2) we set $F = \begin{pmatrix} \frac{dx}{dt} \\ \frac{dy}{dt} \end{pmatrix} = \begin{pmatrix} y \\ -\gamma y - \omega^2 \sin(x) \end{pmatrix}$ and 2D-Taylor expand around arbitrary critical point (x_0, y_0) :

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(2) we set $F = \begin{pmatrix} \frac{dx}{dt} \\ \frac{dy}{dt} \end{pmatrix} = \begin{pmatrix} x - y \\ -\gamma y - \omega^2 \sin(x) \end{pmatrix}$ and 2D-Taylor expand around arbitrary critical point (x_0, y_0) :

$$\begin{aligned} F(x, y) &= F(x_0, y_0) + J_F(x_0, y_0) \begin{pmatrix} x - x_0 \\ y - y_0 \end{pmatrix} + O(\|(x - x_0, y - y_0)\|^2) \\ &= \begin{pmatrix} 0 & 1 \\ -\omega^2 \cos(x_0) & -\gamma \end{pmatrix} \begin{pmatrix} x - x_0 \\ y - y_0 \end{pmatrix} + O(\|(x - x_0, y - y_0)\|^2). \end{aligned}$$

Presenting the method: oscillating pendulum

- (3) The linearization around $(x_0, y_0) = (n \cdot \pi, 0)$ for even integer n (the downward equilibrium position) is:

$$\frac{d}{dt} \begin{pmatrix} x \\ y \end{pmatrix} = \begin{pmatrix} 0 & 1 \\ -\omega^2 & -\gamma \end{pmatrix} \begin{pmatrix} x - n \cdot \pi \\ y \end{pmatrix} + O(\|(x - n \cdot \pi, y)\|^2).$$

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- (4) The eigenvalues of that matrix are:

$$\lambda_1, \lambda_2 = \frac{-\gamma \pm \sqrt{\gamma^2 - 4\omega^2}}{2}.$$

Presenting the method: oscillating pendulum

If $\gamma^2 - 4\omega^2 > 0$, then the eigenvalues are real, distinct and negative. Therefore, the critical points will be stable nodes.

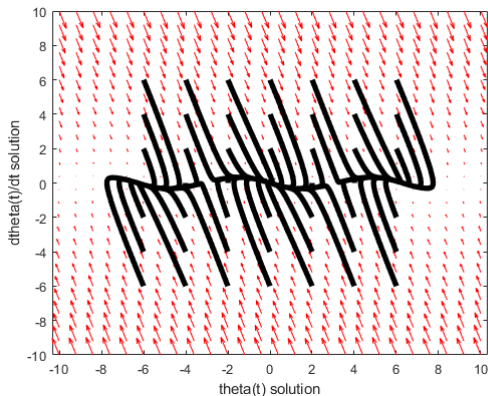


Figure: Stable nodes at even integer n critical points $(n\pi, 0)$ for $n=0,2,-2$.

Presenting the method: oscillating pendulum

If $\gamma^2 - 4\omega^2 = 0$, then the eigenvalues are repeated, real and negative. Therefore, the critical points will be stable nodes.

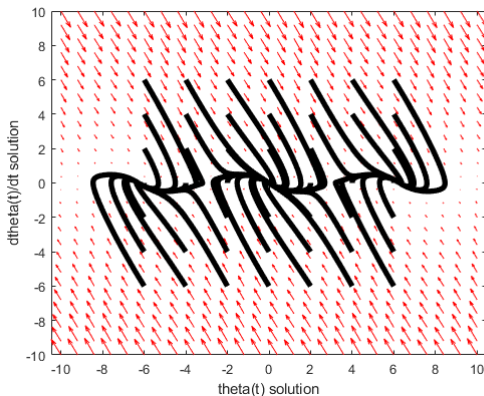


Figure: Stable nodes at even integer n critical points $(n\pi, 0)$.

Presenting the method: oscillating pendulum

If $\gamma^2 - 4\omega^2 < 0$, then the eigenvalues are complex with negative real part. Therefore, the critical points will be stable spiral sinks.

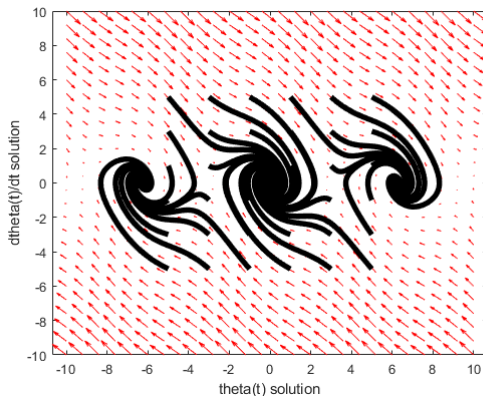


Figure: Stable spiral sinks at even integer n critical points $(n\pi, 0)$.

Presenting the method: oscillating pendulum

The linearization around $(x_0, y_0) = (n \cdot \pi, 0)$ for odd integer n is:

$$\frac{d}{dt} \begin{pmatrix} x \\ y \end{pmatrix} = \begin{pmatrix} 0 & 1 \\ \omega^2 & -\gamma \end{pmatrix} \begin{pmatrix} x - n \cdot \pi \\ y \end{pmatrix} + O(\|(x - n \cdot \pi, y)\|^2).$$

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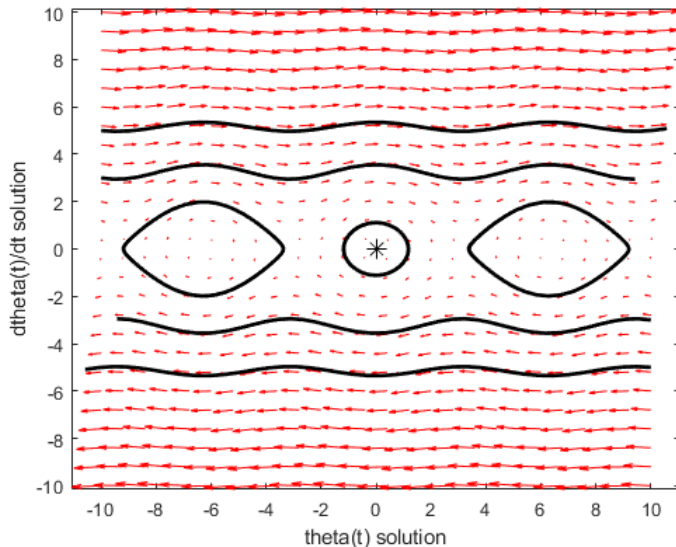
$$\frac{d}{dt} \begin{pmatrix} x \\ y \end{pmatrix} = \begin{pmatrix} 0 & 1 \\ \omega^2 & -\gamma \end{pmatrix} \begin{pmatrix} x - n \cdot \pi \\ y \end{pmatrix} + O(\|(x - n \cdot \pi, y)\|^2).$$

The eigenvalues of that matrix are:

$$\lambda_1, \lambda_2 = \frac{-\gamma \pm \sqrt{\gamma^2 + 4\omega^2}}{2}.$$

Therefore, it has one negative eigenvalue $\lambda_1 < 0$ and one positive eigenvalue $\lambda_2 > 0$, and so the critical points will be unstable saddle points.

Presenting the method: oscillating pendulum



In class example

Consider the system (Duffing's equation)

$$\frac{dx}{dt} = y, \frac{dy}{dt} = -x + \frac{x^3}{6}.$$

- Find critical points and linearize.
- identify stability behaviour.

In class example

Consider the system

$$\frac{dx}{dt} = y, \frac{dy}{dt} = x + 2x^3.$$

- Find critical points and linearize.
- identify stability behaviour.
- find implicit solution.

The End